# DESIGN OF AN ELECTRO SYSTEM FOR PACKAGING SEALING PROCESS IN DAIRY FACTORY " EI LABRADOR " 2015-2016

Richar Arturo Recalde Guerra Technical University North Engineering in Electrical Maintenance richar-90@hotmail.com

*Abstract.* The present work was made with the purpose of designing an electro-pneumatic system for a machine prototype test sealed packaging for dairy factory " El Labrador ", commanded by a PLC and will be implemented in the Laboratory of the Engineering in Electrical Maintenance; in order to improve methods of student learning in the field of industrial control, so that further expand their knowledge in different areas of study and knowledge gained in the classroom implement them performing laboratory tests on the prototype

### I BACKGROUND

The creamery " El Labrador " is located in the province of Carchi, canton Mirror, parish of San Isidro. This aims at the development and distribution of dairy products, and it makes packaging processes and sealing of milk and its derivatives.

In this factory, the process of sealing containers of yogurt is essential to ensure proper conservation and distribution of products. However , the techniques used for this process do not have an automated system to avoid wasting time and to dispatch all orders placed , a process that helps supply the requirements of the products in the shortest possible time with high quality and production efficiency

#### A. General Goal

Designing a training module for an electro-pneumatic system that will uncover the optimization of the production process of sealing containers of yogurt in the dairy factory " El Labrador "

### B. Justification

Currently pneumatic systems are a great help for process automation in industries. The air within industrial automation has become a very useful tool. Through pneumatic components, it is given simpler, more profitable and future industry application solutions.

By using this type of technology which can be operated by the person who is responsible and allocates a certain area, help improve production in less time. This project will be aimed at improving the factory so that help optimize time, performing sealing work more driveability and a high degree of quality.

# II DIAGNOSIS OF CURRENT SITUATION OF THE FACTORY

The company currently performs the work of sealing the packages manually or traditional way, which leads to production is not greater than the defendant.

#### Table 4

	Production	Time	Time
Liter	40	daily	15
containers			minutes
Small packages	100	daily	1 hour
Small packages	500	weekly	5 hours

Sealed data processed products not automated.

### III SELECTION OF ELEMENTS ELECTRICAL, ELECTRONIC, TIRES, AND MECHANICS.

#### A. Module Operation

1) The conveyor belt performs the transport of containers in a continuous and regular basis.

2) When the sensor located opposite the cylinder B detects the presence of container, this cylinder pushes the container to the sealing position.

3) When the package is located in the proper sealing position sensor cylinder A perform the sealing process while the cylinder B is retracted to advance the sealed container.

4) The sealed container and advances from the side until the cylinder C sensor push it out for a rail where all already sealed containers for final review will be stored.

### B. Design parameters of the conveyor belt

- Diameter transport container
- Band length
- Productivity System
- Diameter of drums
- Maximum weight per package
- Number of packages within the band

For the construction of the conveyor belt iron material was used for part of the structure, also as for mechanical parts and bearings.

It has been made perforation a part of the structure horizontally so serves as guides for tuning the conveyor.



Figure 1: Drilling to adjust the band

As illustrated here is building one of the rollers of the conveyor, the same will have a longer axis of which is coupled to a pulley and a rotating side which will be coupled to the motor. The other roller is so performed in the same way with the difference that their axes are the same length, this roller will be the anger in the space where the guide conveyor belt adjustments were made.





As shown in figure 3, before attaching the bearings to the structure proceeded to conduct verification with each of the rollers to avoid any inconvenience later.



Figure 3: Check roller shafts for bearings

Once it verified that all elements are in perfect condition they proceeded to assemble the bed of the conveyor belt



*Ilustración 4:* Ubicación de elementos en estructura principal

Finally, we proceeded to locate the nylon band in the structure and to make the respective settings of each of its elements.



Figure 5: Location of the conveyor belt

C. Selection of electrical and electronic pneumatics

# Selecting the cylinder B

To select the cylinders was taken into account that the first will be to push the package through the sealing position, with this in mind that will need to calculate a required clamping force for both

### Table 5

Experimental data for calculating the cylinder B

Pressure	6 (Bar)	
Strength	unknown	
Frictional force	90%	
Diameter	25mm	

$$F = P.A = 10. p. \pi \left(\frac{d^2}{4}\right). n$$

We replace values

F = 10. (6|bar|). 
$$\pi \left(\frac{(2.5 \text{cm})^2}{4}\right)$$
. (0.9)  
F = 265.07 (N)

With the strength obtained 265.07 [N] can select a cylinder with a diameter of 2.5 [cm] or 25 [mm] at a pressure of P = 6 [Bar] and 265 F = force [N] and one carrea 80 [mm].

En la siguiente tabla se detallan el modelo del cilindro:



Figure 6: Double-acting cylinder B.

### Table 6

Specifications Cylinder B

Technical specifications		
Kind	Double acting Ø 32	
	200 mm	
Material	Aluminium alloy	
Temperature	0°C a 70°C ~ 0°F a	
	158°F	
Work pressure	1 a 9 Bar ~ 15 a 130	
	PSI	
Cushioning	Pneumatics, dimmable	
Fluid	Filtered compressed	
	air, lubricated or non-	
	lubricated	

#### Selecting the cylinder C

The same calculation is performed taking into account that this will be in charge of taking the conveyor belt to the rail sealing storage container, the actuator must be greater than the previous race.

### Table 1

Experimental data for calculating the cylinder C

Pressure	6 (Bar)	
Strength	265 (N)	
Frictional force	90%	
Diameter	25mm	

$$F = P.A = 10. p. \pi \left(\frac{d^2}{4}\right). n$$

Punting d we have:

$$d = \sqrt{\frac{4F}{10p. \pi. n}}$$
$$d = \sqrt{\frac{4(265)}{(10)(6)(\pi)(0.9)}} = 2,49 \text{ [cm]}$$

With the diameter obtained we can select I cylinder we will use, with a diameter of 2.49 [cm] or 24.9 [mm] at a pressure of P = 6 [Bar] and a force F = 265 [N] and career 200 [mm].



Figure 7: Double-acting cylinder C

### Table 2

Specifications Cylinder C

Especificaciones Técnicas		
Kind	Double acting Ø 32	
	200 mm	
Material	Aluminium alloy	
Temperature	0°C a 70°C ~ 0°F a	
	158°F	
Work pressure	1 a 9 Bar ~ 15 a 130	
	PSI	
Cushioning	Pneumatics, dimmable	
Fluid	Filtered compressed	
	air, lubricated or non-	
	lubricated	

### Selecting the cylinder A

For the selection of this cylinder the following aspects will be taken into account, this cylinder will be in charge of making the covered container, taking into account that should be enough pressure to put the lid in place.

#### Tabla 3

Experimental data for calculating the cylinder A

Pressure	6 (Bar)
Strength	270 (N)
Frictional force	90%

d

$$F = P.A = 10. p. \pi \left(\frac{d^2}{4}\right). n$$

Punting d we have:

$$d = \sqrt{\frac{4F}{10p. \pi. n}}$$
$$= \sqrt{\frac{4(270)}{(10)(6)(\pi)(0.9)}} = 2,52 \text{ [cm]}$$

With the diameter obtained we can select I cylinder we will use, with a diameter of 2.52 [cm] or 25.2 [mm] at a pressure of P = 6 [Bar] and a force F = 270 [N] and career 80 [mm].



Figure 8: Double-acting cylinder A

# Table 4

Specifications Cylinder A

<b>Technical specifications</b>		
Kind	Doble efecto Ø 32	
	200 mm	
Material	Aleación de Aluminio	
Temperature	-5°C a 70°C ~ 23°F a	
	158°F	
Work pressure	2 a 9 Bar ~ 28 a 130 PSI	
	resilient	
cushioning		
Fluid	Filtered compressed air,	
	lubricated or non-	
	lubricated	

### Selecting maintenance unit

To choose the maintenance unit each of the elements to choose from and installed according to the specifications we analyzed a FRL, filter-regulator lubricator was chosen.



Figure 9: Maintenance unit

### Solenoid valves

The selection of the solenoid valves is performed using the data obtained when the dimension of the cylinders to be used, for which the solenoid valves 5/2 monostable type were chosen.



Figure 10: Solenoid valves 5/2 monostable

#### Table 11

Technical Data for valve 5/2

Technical specifications		
Model	5/2 solenoid valve	
Operating pressure	0-8 Bar ~ 0-114 PSI	
Fluid	Filtered compressed air, lubricated or non- lubricated	
Work temperature	-5 ° C to 60 ° C ~ 23 ° F to 140 ° F	
Answer time	0.05 seconds	
Maximum cycling	5 Cycles / second	
Material	Aluminum	

#### Sensors

The sensors were chosen to occupy the network switch type sensors for its versatility, easy installation and cost. The work of this sensor is detecting the position of the pneumatic cylinder rod during operation of the machine.

Similarly photoelectric sensors which are going to be responsible for detecting the presence of the package during the sealing process was chosen.



Figure 11: Sensor reed switch

# Table 12

Technical	specifications	of reed	switch sensor	

Technical specifications		
Contact	Normally open	
Indicator	led top	
Temperature	0 to 60 ° C ~ 32 to 140	
	°F	
Protection	IP 67	
Voltage Range	5 ~ 380V AC , 240V	
	DC 5 ~	
Current Rating	AC 5 ~ 50 mA , 60 mA	
	DC 5 ~	
Vibration Resistance	10 ~ 50 Hz	
Answer time	1ms	
Cable length	2 Mts	



Figure 12: Photoelectric Sensor

# Table 13

Technical specifications of the photoelectric sensor

Technical specifications		
Kind	50x50 compact plastic	
	body	
Diffuse Range (m)	1	
Scale polarized reflex (	4	
m) (With reflector		
accessory)		
Scale background	0.4	
suppression (m)		
Through-beam (m)	twenty	
(with accessory)		
Dimensions (mm)	SS 50x18x50	
Temperature	Relay 50x18x50	

# Pipeline

In this part the selection of the pipes performed taking into account the connectors to use with cylinders and other actuators type, making it proceeded to elect the hose in polyurethane. This type of hose for its flexibility, its duration, its weight q is very light and is ideal for pneumatic connections was chosen.



Figure 13: Tubería

### Table 14

Technical data pipe

Technical specifications		
Fluid	Compressed air	
Work pressure	0 to 10.3 Bar ~ 0-150	
_	PSI	
Vacuum pressure	-750mm Hg (10Torr)	
Work temperature	$0 \circ C$ to $60 \circ C \sim 32 \circ F$	
_	to 140 ° F	

# Fittings and mufflers

As the pipe is to use a hose fittings <sup>1</sup>/<sub>4</sub> millimetric be used straight union, union elbow, union T, depending on the connection we wish to do likewise air mufflers were selected.



Figure 15: Fittings

### Tabla 5

Especificaciones para los racores

<b>Technical specifications</b>			
Fluid	Compressed air		
	0 a 10.3 Bar ~ 0 a		
Work pressure	150 PSI		
	-750mm Hg		
Vacuum pressure	(10Torr)		
	0°C a 60°C ~ 32°F		
Work temperature	a 140°F		



Figure 16: Mufflers

# Tabla 6

Specifications for mufflers

Technical specifications			
	Compressed air		
Fluid			
	0 a 10 Bar ~ 0 a 142		
Work pressure	PSI		
Vacuum pressure	zinc plated bronze		
	0°C a 70°C ~ 32°F a		
Work temperature	158°F		

#### **Compressor selection**

For compressor selection was guided in the different parameters were obtained by choosing the different cylinders which will be responsible for the job.



Figure 18: Compressor

### Tabla 7

### Technical data compressor

Especificaciones Técnicas			
Voltage	110 V		
Frequency	60 Hz		
Engine	ne 550 W – HP: 0.75		
Air Production	70 l/min		
Air pressure	0.75 Mpa		
Container volume air.	32 L – 7 Gal.		
Noise	56-65 dB(A)		
Weight	30 Kg		
Dimensions	56x56x70 cm		

### Selection and sizing of conductors protection

To make the design of the electrical conductors we will build on the engine power that would be the biggest load within the system, for which tables specified electrical conductors where both current, permissible operating voltage and temperature were used.

For this we use the equation of electric power:

mechanical power

$$P = V.I$$

Where power P = W

Electrical power

$$W = V.I$$

Then as data have the engine power that is <sup>1</sup>/<sub>4</sub> HP and voltage is 220 V will replace these values in equation 9 to calculate the current and determine the type of driver.

$$I = \frac{186.425 \, W}{220 \, V} = 0.846 \, A$$

The following table shows the type of conductor according to the allowable operating temperature specified.

### Tabla 8

Calibers electrical conductors

Amperaje que soportan los cables de cobre					
Nivel de temperatura:	60°C	75°C	90°C	60°C	
Tipo de aislante:	TW	RHW, THW, THWN	THHN, XHHW-2, THWN-2	SPT	
Medida / calibre del cable	Amperaje soportado			Medida / calibre del cable	Amperaje soportado
14 AWG	15 A	15 A	15 A		
12 AWG	20 A	20 A	20 A	20 AWG	2 A
10 AWG	30 A	30 A	30 A		
8 AWG	40 A	50 A	55 A	18 AWG	10 A
6 AWG	55 A	65 A	75 A	TO AWG	
4 AWG	70 A	85 A	95 A		
3 AWG	85 A	100 A	115 A	16 AWG	13 A
2 AWG	95 A	115 A	130 A		
1 AWG	110 A	130 A	145 A		18 A
1/0 AWG	125 A	150 A	170 A	14 AWG	
2/0 AWG	145 A	175 A	195 A		
3/0 AWG	165 A	200 A	225 A	12 AWG	25 A
4/0 AWG	195 A	230 A	260 A		

#### Tabla 19

Conductors and electrical protection

Protection and	Protection and conductor sizes		
Schneider electric	110 V a 2 A		
breaker			
Schneider electric	220 V a 6 A		
breaker			
Contactor	24 VDC		
Relays	24 V		
Fuses	2 A		
Guard engine	1.6 a 2.5 A		
Power supply	24 V		
Conductors	3x12 AWG concentric		
	conductor		
	Conductor # 22 AWG		



Figure 20: Internal Control Panel view

## IV VALIDATION MODULE DESIGN

### A. Determination parameters

The parameters were set times and number of sealed containers, these data were taken with the training module already in operation stage.



Tabla 20

Comparison of results					
Validation of the training module					
Before		After			
Minutes	15	40	Minutes	1.10	6
Hours	1	100	Hours	1	327
Weekly	5	500	Weekly	5	1635

A. Diagrama de fuerza



# B. Diagrama de bloques



# **V** CONCLUSIONS

With the construction of an automated packaging module sealing it is shown to be possible to automate this process on an industrial scale.

Automation package sealing process leads to increase the production capacity of the factory, by reducing processing time and reducing losses or waste default.

The implementation of the training module, will be very useful to expand and strengthen the knowledge of students of the School of Engineering in Electrical Maintenance in the area of electro-pneumatic control.

### VI RECOMMENDATIONS

It is recommended that a periodic review of each element of the module, prior to its implementation to avoid possible damage during operation of the machine, keep in good condition and preserve the module sensors and pneumatic actuators.

Students would be wise to seek ways to expand the module by implementing new sensors or actuators.

Before operating the module and make changes should read the manuals for each of the equipment and programming manual developed in this paper grade to ensure proper operation of the instructions that are programmed into the PLC.

#### REFERENCES

- Agudelo Alvarez, R. (22 DE FEBRERO DE 2012). Automatización Industrial.
- [2] Anónimo. (28 de abril de 2011).Electroneumática.
- [3] Aragonés, O. B., Saigí Grau, M., & Ferran Zabaleta, A. (1993). Automatismo Eléctricos Programables. Barcelona.
- [4] Barreto Véliz, B. P. (s.f.). *Controladores Programables*.
- [5] Bernal, V. H. (2010). Automatización. *Guia de Trabajo*.
- [6] Bolton, W. (2006). Programmable Logic Controllers.
- [7] Calderón, J. J. (2011). *Electroneumática*. Volúmen 01.
- [8] Carmona Preciado, J., & Perez Arango, J. A.
  (2014). Diseño electroneumático para maquina de doblado y planchado de prendas. Pereira.
- [9] Casignia Vásconez, B., & Gavilánez Carvajal, H. J. (2011). Diseño, implementación de un módulo didáctico para simulación de procesos industriales en una banda transportadora, por medio de PLC. Riobamba.
- [10] Castiñeira, N. H. (2010). Sistemas Nuemáticos.
- [11]Catalogo BKB. (2015). Maquinaria Industrial, 16-17.
- [12] Creus Solé, A. (2007). Neumática e Hidraulica.
- [13] Creus Solé, A. (2011). Neumática e Hidraúlica. Barcelona: Alfaomega.

- [14] Domingo Peña , J., Martínez García, H., & Gamíz Caro , J. (2003). Introducción a los autómatas programables.
- [15]Eggel, R. D., Fernández, J. P., & Killer, C. (2013). Introducción a la Electroneumática.
- [16] Escalera Tornero, M. J., & Rodríguez Fernández, A. J. (2009). Actuadores Neumáticos.
- [17] Escalera Tornero, M. J., & Rodríguez Fernández, A. J. (2012). Actuadores Neumáticos.
- [18] Escalona Moreno, I. (2014). Circuitos Neumáticos y Aplicaciones de Ingeniería Industrial.
- [19] Fernández Amador, G. (2005). Sensores Magnéticos e Inductivos.
- [20] Festo. (2012). Electroválvulas, Válvulas Neumáticas, Midineumática.

### SOBRE EL AUTOR

**Author:** Richard A. G. Recalde, completed his studies at the Engineering Degree in Electrical Maintenance Technical University North of the city of Ibarra, Imbabura, Ecuador. (2010-2015).